

## Eco-Friendly Brand New Energo-Technological Complexes: Ensuring Innovation-Driven Development Of Organic Energy Industry

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### Abstract

To date, concerns have been increasingly raised about emissions of greenhouse gases (CO<sub>2</sub>, in particular) and global warming. Oxy-fuel combustion constitutes one of the key technologies to curb emissions of CO<sub>2</sub> in the energy sector. The units with pure oxygen-enriched fuel combustion are attractive due to a minimum content of harmful admixtures in the oxy-fuel combined cycle gas turbine units (OCCGTUs) exhaust gas. Waste fluid therewith contains mostly vapours of water and carbon dioxide, and only CO<sub>2</sub> remains after water condensation.

For a unit with oxy-fuel combustion, the compressor-less combined cycle gas turbine unit is considered, where, in the liquid phase, pumping equipment increases pressure of all fluid components prior to their supplying to a combustion chamber. There

is a description of the process flow diagram (PFD) of the suggested unit that employs organic Rankine cycle (ORC), which substantially augments (by about 5 absolute percent) the unit efficiency factor (EF). Potential advantages of compressor-less combined cycle gas-turbine unit have been presented.

The article examines the prospects of fusing the suggested PFD in energy-technological complexes of different types. The "Arctic Cascade" Project with compressor-less combined cycle gas turbine unit and integration of the power-producing unit thermodynamic cycle with the liquefied natural gas plant (LNG-plant) bring considerable economic and environmental benefits.

It is necessary to widen the searches for innovative concepts of creating high-efficiency power-producing units to the systems, that take into account energy production preparation, and energy generation, potential combination with process cycles and manufacture of chemical products, and utilisation of production waste as well.

**Keywords:** carbon capture, oxy-fuel combustion, compressor-less combined cycle, organic Rankine cycle, air separation unit, liquefied natural gas

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## 1. Introduction

It is beyond question that creation of zero-carbon global economy will attain a key position for the nearest decades. The recognised principal aspects of this task involve climate and ecology, and their interrelation and influence on the biosphere have become increasingly prominent every year. More and more scientists, politicians, and public organisations state that there is a need for a radical transition of human activities to eco-climatically safe technologies. The entire cycle of activities is considered therewith, as the saying goes: "from here, taking all side effects into account". Now, the end product price index also includes "natural" consequences of its production, the whole lifecycle, involving utilisation. Red indicators of these indices become the most important aspect of not only moral but legal prohibition of using the entire number of technologies habitual for us, what is most pronounced in power sector.

It is commonly known that electric power industry is multifaceted and diverse. Thermal, helio-, nuclear, wind, hydro-, hydrogen, emission, chemical, autonomous and integrated, small, big, local, green, etc. types are found in the literature. In recent years, the adjectives "renewable" and "green" are considered a compliment, and "thermal" sounds abusive, an assault on the ears. A sustained note in mass media of the early 21<sup>st</sup> century: solar batteries are praised, and the plans to close coal and nuclear power plants are reported. A swift decline in organic fuel-based energy industry has been predicted. Carbon dioxide emission – greenhouse gas, greatly affecting the Earth's climate, is the major argument here. Legislative initiatives to ban not only generation of electricity using organic fuel, but also its purchase, have been introduced. From 2022, Europe plans to introduce a cross-border levy for carbon imports. More and more electric energy has been produced from renewable sources, with prevailing percentage of this generation through installing solar arrays on the roofs of residential, commercial and industrial buildings, i.e., small and medium, rather than large solar power stations, occupying vast land areas. According to the International Energy Agency (IAE), even by 2025 the percentage

of renewable energy sources (sun, wind, water, and biofuel) in global electrical energy generation will reach 30% [1].

Still, is everything so unambiguous with no options in the problem of using in power sector the organic energy resources, Russia is extremely rich in?

First and foremost, we shall recall some features of and conventional approaches to using organic fuel in power sector. In Russia, as in most countries of the world, electric power plants on organic fuel tend to predominate nowadays. The absolute value of CO<sub>2</sub> emission by these plants, equivalent to about a quarter of all man-made emissions of carbon dioxide gas, is related to fuel combustion in thermal machinery, and depends on the fuel calorific value and the efficiency of the machinery itself. Thus, comparing coal and hydrocarbon methane, the latter is obviously more advantageous owing to a higher calorific value. In addition, there is less carbon in methane, and water is known to be produced as a result of the methane hydrogen oxidation reaction. In this context, pure hydrogen has an even greater advantage as a fuel. Although, taking into account the expenses incurred during the hydrogen production cycle and its storage, these advantages are then ambiguous.

### 1.1. Efficiency indicators of power-generating unit

There is also no unambiguous answer in the case of thermal machinery. Here, the efficiency of electric generating unit is defined by numerous factors. Combustion of natural gas in combined cycle gas turbine units (CCGTUs) is known to be the most cost-effective. However, it should be made clear as well. Hence, in Russia CHPPs with considerable cogeneration loads (mean relative ratio between centralised heat generation and electric energy is 1.46) are frequent. "Classic" CCGTU–gas-turbine unit with steam-turbine Rankine cycle waste-heat loop, employed at the CHPP, offers no optimal solution. Using "classic" CCGTU as a power unit for some technological complexes of chemical industry is also not optimal [2].

It should be mentioned that the efficiency factor (EF), a widely recognized indicator of the efficiency of thermal equipment, is not its absolute characteristic. EF defines the equipment capability for converting the fuel chemical energy to mechanical energy; a considerable portion of heat energy conversion is left out. Further actions for converting the energy, produced by the equipment, to the required electricity, or other net energy have also been left aside. Thus, in case of the above-mentioned cogeneration CHPP mode, there are characteristics, that make EF more precise, more exactly defining the efficiency of the equipment operated in heat-end mode (1, 2), and the units themselves, which ensure such a consumer's duality with maximum efficiency, are called cogeneration units.

$$CFU = (N_e + Q_t) / H_t, (1)$$

where:

CFU – coefficient of fuel utilisation;

$N_e$  – effective electric capacity;

$Q_t$  – supplied thermal capacity;

$H_t$  – total energy of consumed fuel.

$$\eta_e = N_e / (H_t - Q_t / \eta_b), (2)$$

where:

$\eta_e$  – EF of heat-end mode electric energy generation;

$N_e$  – effective electric capacity;

$Q_t$  – supplied thermal capacity;

$H_t$  – total energy of consumed fuel;

$\eta_b$  – EF of heating boiler.

CFU and EF are very informative for multipurpose plants, which produce extra market products in addition to electric energy. CFU demonstrates high efficiency of cogeneration and trigeneration as compared to separate production of electric energy and the very same heat and cold. In Russia, cogeneration power units have been rather extensively applied. There are projects of cogeneration CCGTUs with a very high efficiency, due to high values of initial cycle parameters and potential use of low-temperature heat of flue gases, including condensation heat of water, formed during fuel combustion [3-6]. Of note also is a recent growing interest in trigeneration units, which provide complex generation of electricity, heat, and cold. From the middle of the past century, similar to them integrated energy-technological complexes have been used in chemical industry. Nowadays, cogeneration and trigeneration projects have been offered for gas industry, which allow organic fuel to be used fairly efficiently, and CO<sub>2</sub> greenhouse gas emission to the atmosphere to be thus reduced [7].

Even so, with all obvious efficiency of mentioned cogeneration and trigeneration units, the problem of carbon dioxide sequestering for them still remains unsolved. The attempts to create a device capable of efficiently capturing CO<sub>2</sub> from the heat engine exhaust gases (Post-combustion capture - PostCC) are still unsuccessful. Here, of importance are the costs of the complete end-to-end PostCC cycle, including production of liquid CO<sub>2</sub>, convenient for storing, transporting, using, and burying [8]. A system of removing CO<sub>2</sub> prior to combustion (so called system of Pre-combustion Capture) can provisionally facilitate solving this problem. The process implies methane conversion to the previously mentioned hydrogen and carbon dioxide, with further extraction and combustion of pure hydrogen in the thermal equipment. Abroad, such treatment is estimated at 60...30\$ per a tonne of CO<sub>2</sub> [9], although these numbers do not include further expenses associated with gaseous state of carbon dioxide.

## 2. State of the art

Surely, there is no point in analysing the entire range of possible combinations of devices in power-producing units in terms of CO<sub>2</sub> sequestering. Only the most promising solutions will be mentioned.

The last decades have seen an intensification of works on the CCGTU projects with combustion of organic fuel in oxygen (OCCGTU). In the 21<sup>st</sup> century, the research and practice development of the formerly known anaerobic (air-independent) power units of submarines has been underway in the land-based power industry as the units that ensure a complete capture of CO<sub>2</sub> [10, 11]. One of the most frequent in the literature descriptions of how such OCCGTU function, is a so-called thermodynamic Allam cycle, based on which first power stations in USA have already been built [12]. These units are attractive because of minimum content of harmful admixtures in the OCCGTU exhaust gases.

Waste working fluid there with contains mostly vapour of water and carbon dioxide, and only CO<sub>2</sub> (provided that there is no sulphur and other harmful admixtures in the primary fuel) remains after water condensation. A required amount of CO<sub>2</sub>, withdrawn from exhaust gases, is supplied to the combustion chamber of the unit with such thermodynamic cycles, in addition to organic fuel and oxygen, as a dead matter, to create an acceptable for the turbine gas temperature. In this context, the Allam flow diagram of OCCGTU largely resembles a closed-cycle unit. One option of such solutions implies the increase in pressure of all fluid components prior to their supplying to the combustion chamber in the liquid phase by pump equipment. Such OCCGTUs, in which the pressure of fluid is increased in the liquid phase, without compressors, are called compressor-less [13].

The cycles of compressor-less OCCGTU have several considerable advantages that serve as prerequisites for their extensive use in the near future. The possibility of most effectively sequestering carbon dioxide is certain to be the main factor that will facilitate the implementation of compressor-less cycles. Hazardous emissions will be almost excluded, what will make it possible to remove CO<sub>2</sub> and H<sub>2</sub>O combustion products from the cycle in the liquid state. The water is of rather high quality to be used with minimum water treatment in various technologies or feed the heating networks. Carbon dioxide from the compressor-less OCCGTU is brought out in a form, the most convenient for transportation, including transportation by pipelines, due to extremely low viscosity of CO<sub>2</sub> [14]. Compressor-less cycles offer quite as well prospects of reaching the electric energy production efficiency factors as the best modern CCGTUs [13]. When heat along with electricity are produced, not only all heat by the low heat value of fuel, but a substantial portion of vaporisation heat of water, formed as a result of combustion, are used here [15].

Compressor-less cycles belong to the category of semi-closed. Retaining the major advantage of open cycles (intra-cycle combustion and, thus, few energy losses when supplying heat from a hot source through the heat-exchange surface), they have virtually all advantages of closed cycles, with the possibility to efficiently control the power unit capacity as one of them [16]. Maintaining principal thermodynamic parameters (expansion rate in the turbine, temperatures of hot and cold sources) and changing pressure in the combustion chamber, through changing the working fluid rate, the OCCGTU capacity changes with almost no changes in the thermodynamic cycle efficiency factor.

Since compressor-less OCCGTU has no air compressor (exclusive of means to produce oxygen, in case of using air separation unit (ASU)), and fuel and oxygen can be used in the liquid form, a cycle with high initial pressure, hence, with high specific capacities of turbines will be implemented. Such an opportunity will enable a substantial reduction in the OCCGTU weights and dimensions, and a decrease in capital expenditures for the overall construction of the entire power station.

An important advantage of compressor-less thermodynamic cycles also involves a possibility of independently controlling the production of thermal and electric energy in a wide range. The compressor-less OCCGTU, efficiently operated in summer, in winter period can meet all demand for heat during the coldest days with no additional peaking boiler plants. As opposed to classic CCGTUs, compressor-less OCCGTU in cogeneration mode per each kW of electric capacity can produce 4 kW of thermal capacity. It will allow Russia to discard additional peaking boiler plants, what will reduce capital expenditures for CHPP construction and decrease operating expenses, including the expenses to capture CO<sub>2</sub> beyond boilers.

The OCCGTU efficiency with allowance for the costs incurred on oxygen production is high, and the price of generated electric energy does not exceed its cost at the power stations with classic CCGTUs [17, 18]. And the initial state of all components of working fluid in the liquid phase will enable short-term accumulation of their reserve and distribution of energy expenditures for their production and use as to smooth daily load curves of outside energy consumers [19].

The problem raised in the article about optimising the thermodynamic cycle and design layout of electric power station, for eco-friendly purposes, is directly related to the location of its operation. The CCGTUs, available on the market, are usually equipped with gas turbine engines (GTE), initially designed to operate in the middle latitudes, at the compressor-inlet air temperature of 288 K or 300 K. When CCGTU operates in other climatic conditions, the GTE parameters become non-optimal, even with the overall growth in its and the entire CCGTU efficiency factor, when ambient air temperature decreases. Here, if the isotherms of mean annual temperature in Russia are analysed (Fig.1), the vastness can be noted of the territory with temperature conditions, that differ, today and in the foreseeable future (Fig.2). Previously, the constructors of the LNG-plant of the Novatek Company in Yamal responded to this peculiarity and developed

their own project of liquefying natural gas, named “Arctic Cascade”, which takes the specifics of local cold climate into account [20-26].

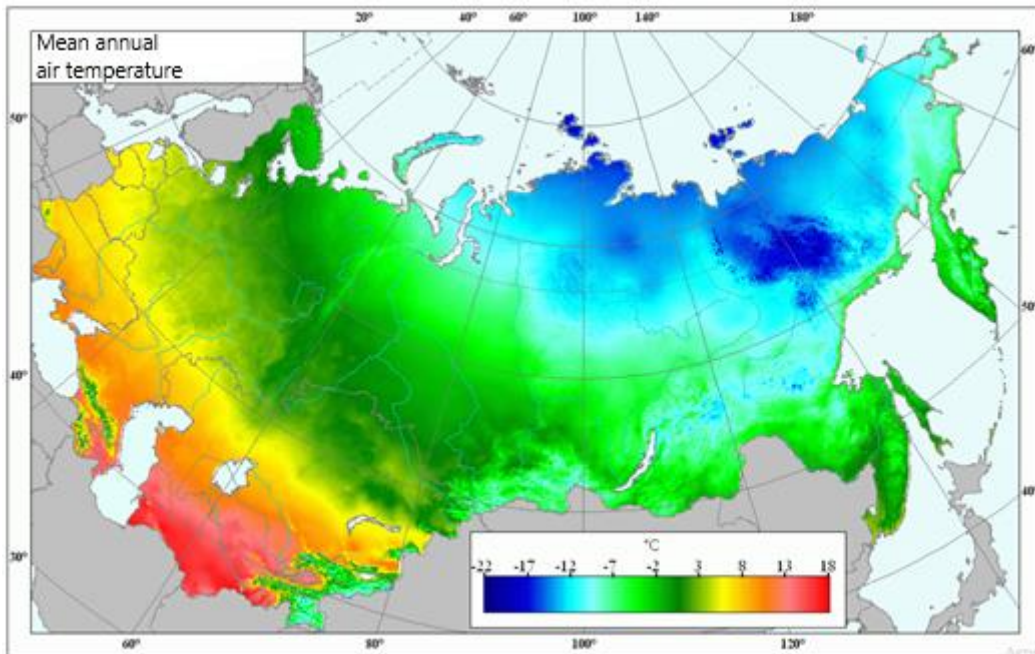


Fig. 1. Mean annual air temperature in the CIS territory [20]

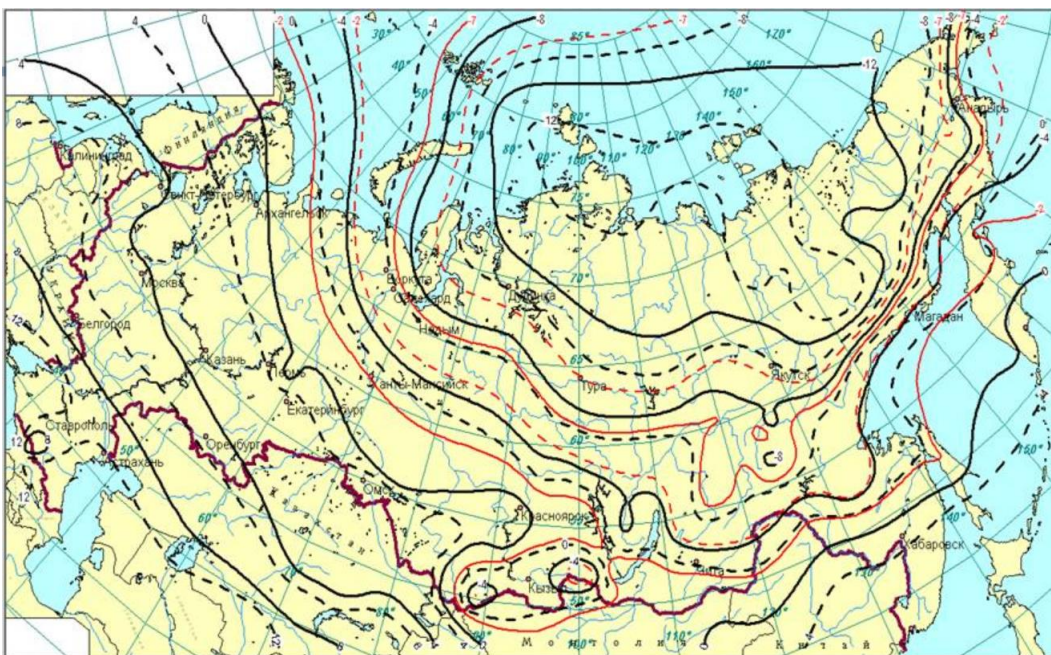


Fig. 2. Forecasted displaced mean annual isotherms by the mid-21<sup>st</sup> century (dashed lines)[21]

In a similar manner, in the thermodynamic cycle of the compressor-less OCCGTU project, taking into consideration a reduced temperature of the cold source – the ambient environment, a

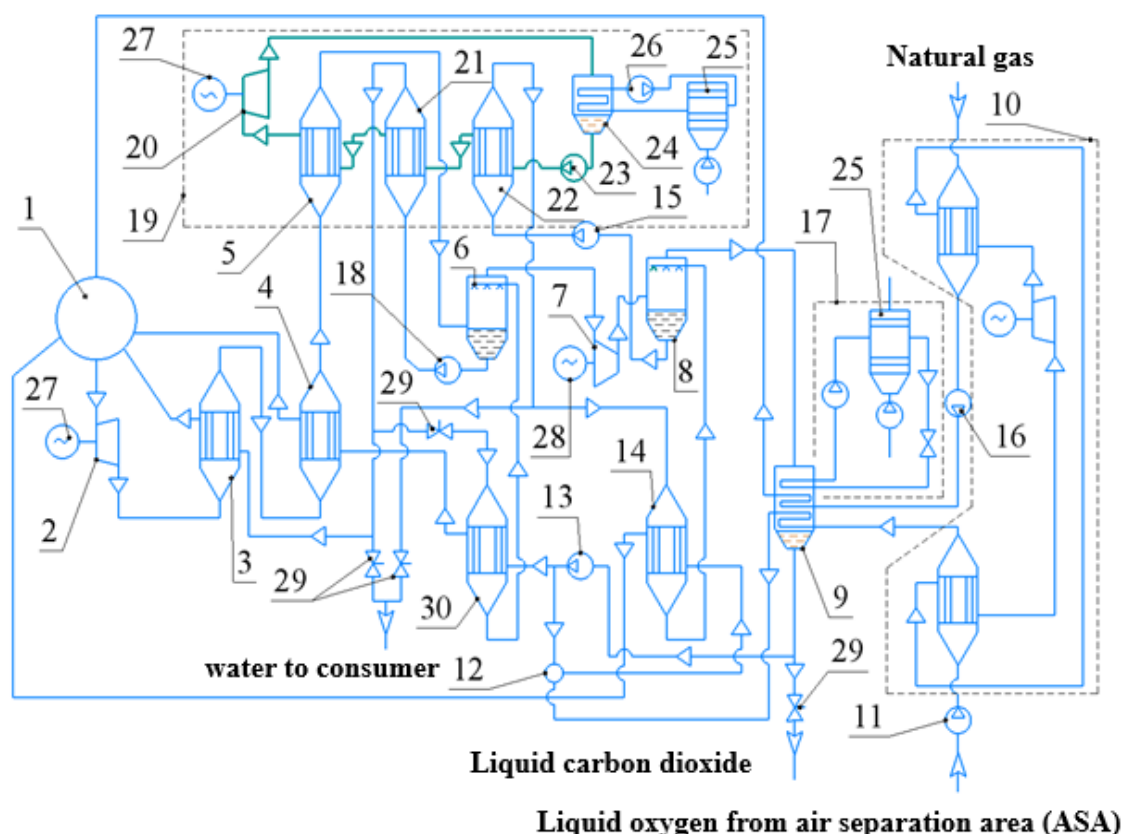


rather considerable increment in the efficiency can be achieved and not only due to cold deficiency reduction in ASU and when liquefying CO<sub>2</sub> [27]. Thus, high temperature of water condensation in the compressor-less OCCGTU cycle and low temperature of cold source constitute optimal conditions for using Organic Rankine Cycle (ORC). In this case, using ORC provides a substantial increment: about 5 absolute percent of the compressor-less OCCGTU efficiency factor.

### 3. Compressor-less combined cycle gas turbine description

Figure 3 presents one of possible diagrams of such binary plant. Main elements of the plant are shown in the figure as positions 1-30, the directions of fluids movements are indicated. Such plant operates according to the following algorithm.

Carbon-containing fuel, for example, natural gas, burnt in the mixture of oxygen, water vapour, and carbon dioxide, is supplied to the combustion chamber (position 1, from this point on: Fig.3). Oxygen there with can be supplied from many known source of oxygen, for instance, can be produced by ASU, incorporated in the power-producing unit and receiving the electricity needed from it.



**Fig.3.** Process flow diagram of compressor-less OCCGTU unit with ORC [28]

1-combustion chamber; 2- combined cycle gas turbine; 3-first cooler of waste gases; 4-second cooler of waste gases; 5-organic steam superheater; 6-first contact cooler of waste gases; 7-compressor; 8-second contact cooler of waste gases, heat exchanger of carbon dioxide cycle heater (of the first cooler of waste gases); 9- carbon dioxide condenser, heat exchanger of oxygen cycle heater, heat exchanger of



natural gas cycle heater; 10- natural gas liquefaction unit;11-oxygen control pump;12-mixing device;13-carbon-dioxide control pump;14-heat exchanger of carbon dioxide and oxygen mixture cycle heater-cooler ofthe secondcontact coolerwatercircuit; 15-circulation pump ofthe secondcontact coolerwatercircuit;16-fuel control pump; 17-unit of the heat pump for discharging heat energyto ambient environment; 18-water control pump; 19-ORC steam-turbine unit; 20-ORC turbine;21-ORC evaporator-water cooler unit of the first contact cooler of waste gases;22-ORCpreheater-cooler of the secondcontact coolerwatercircuit; 23-ORCfeed pump;24-ORC condenser;25-cooling tower; 26-circulation pump;27-turbogenerator; 28-actuator;29-adjustable gates; 30-heat exchanger of carbon dioxide cycle heater.

All activegases, includingcarbon-containingfuel, arecompressedintheliquefied state using pumps, what decreases energy consumed to re-pumpingand reaching the required pressure.

Combustionproductsareexpandedincombinedcyclegasturbine (2) withbackpressuremuch higher than the atmospheric (about 0.9 MPa), andsequentially pass through the first (3), second (4) cooler of waste gasesandorganicsteamsuperheater(5) ofORCunit (19), where waste gases are cooled to the temperature of water condensation onset owing to cooling waste gases to the temperature of at least 420 K. Fromorganicsteamsuperheater (5) cooledwastegasesenterthefirstcontactcooler (6) of waste gases, in which waste gases are cooled to the temperature of at least 273 K, and water in waste gases is condensed. Compressor (7) supplies waste gases from the first contact cooler (6) to thesecondcontactcooler (8) under the pressureclose to 3.5 MPa, where they are cooled to the temperature close the temperature of carbondioxidecondensation onset. Withthat, inthesecondcontactcooler (8) condensation of thewater, remained in waste gases, is also in progress. Then, driedgasfromthesecondcontactcooler(8) enterscarbondioxidecondenser(9), whereCO<sub>2</sub>is condensed through cooling to the temperature of at least 273 Kat the gas pressure of somewhat lower than the specified 3.5 MPa. The temperature and pressureinthecarbondioxidecondenser(9) are determined by the need for reachingthemaximumdegreeofcarbondioxidecapturefromcombustion products without forming thesolid phase, providinga high efficiency factorofOCCGTU.

Watercontrolpump (18) sends condensedwaterfromthefirstcontactcooler (6) to combustion chamber (1) through ORC evaporator (21)and heat-exchangerof water cycle heater, located in the first cooler (3) of waste gases. Aportionofwaterthrough heat-exchangerofcarbondioxidecycleheater(30), is sent to the circuit of the first contact cooler (6)of waste gases. Inthiscase, water, passingthroughheatexchanger (30), is cooled to at least 273 K, excludingthe solid phase.Theremainingportionofwaterissenttoan outside consumer. Waterconsumptionforheat-exchanger(6)andoutside consumeris regulated by gates (29).

Condensed carbon dioxide is poured out of carbon dioxide condenser (9) to be further used outside the unit or to be stored; at that, a certain required portion of liquid CO<sub>2</sub> is sent by carbon dioxide control pump (13) to combustion chamber (1) through mixing device (12) and heat-exchanger of cycle heater of carbon dioxide and oxygen, located in block (14) of cooling water of the second contact cooler (8) of waste gases. Control pump (13) sends another portion of carbon dioxide to combustion chamber (1) through carbon dioxide cycle heater (30) and heat-exchanger of carbon dioxide cycle heater of the second cooler (4) of waste gases.

Liquid oxygen from ASU is first sent by oxygen control pump (11), that enables pressurised oxygen supply to combustion chamber (1), to natural gas liquefaction unit (10), where liquid oxygen is heated due to heat exchange with gaseous carbon-containing fuel. Then oxygen enters the heat-exchanger of oxygen cycle heater, placed in the CO<sub>2</sub> condenser (9), afterwards it is supplied to the mixing device (12) for mixing with carbon dioxide, wherefrom it enters combustion chamber (1) as per the above-described route.

Carbon-containing gaseous fuel at the expense of heat exchange with liquid oxygen is liquefied in liquefaction unit of gaseous carbon-containing fuel (10) through using intermediate heat carrier. Control pump (16) further sends the fuel to combustion chamber (1), through heat-exchanger of carbon-containing fuel cycle heater, located in the CO<sub>2</sub> condenser (9).

Hence, water is condensed through pre-cooling of waste gases and heat transfer in heat exchangers (5, 21, 22) of ORC unit (19) to the low-boiling organic fluid, which can be heated to more than 435K temperature, with its phase transition to condenser (24), through ambient cooling. Such a considerable heat drop on organic turbine (20) considerably increases the electric energy generated by turbo-generators (27) of OCCGTU.

A change in the balance of energy generated by ORC turbine (20) at the constant gas temperature upstream of combined cycle gas turbine (2) can therewith be enabled by changing the productivity of water and carbon dioxide control pumps (18 and 13). In this case, to obtain more energy from ORC, the productivity of water control pump (18) is increased, and to have more electric energy from combined cycle gas turbine (2) as related to the ORC turbine (20) - the productivity of carbon dioxide control pump (13) is increased with simultaneously decreasing water supply to the combustion chamber (1). Thus, in combustion chamber (1) a balance is maintained between inert constituents, which are needed to keep the temperature in combustion chamber (1) within the specified limits. The same procedures take place in the compressor-less OCCGTU with cogeneration circuit (instead of ORC), while switching-over the unit thermal generation.

The described algorithm is not the only possible option. The OCCGTU project with ORC will comprise a major task of determining an algorithm for optimal regulation of the OCCGTU operation with maximum efficiency and provision of necessary capacity, with allowance for

changes in ambient temperature, outside heat consumption, amount of accumulated energy. And here different options are possible for regulation, varying not only water and carbon dioxide consumption in combustion chamber (control pumps 18 and 13), but also operation mode of other OCCGTU elements.

Carbon dioxide is condensed in condenser (9) due to:

- pre-cooling of waste gases in the second contact cooler (8) to the temperature close to the temperature of carbon dioxide condensation onset;
- cooling of waste gases in condenser (9) itself, owing to cycle heating of oxygen and fuel;
- transfer of heat through cooling tower (25) to ambient environment.

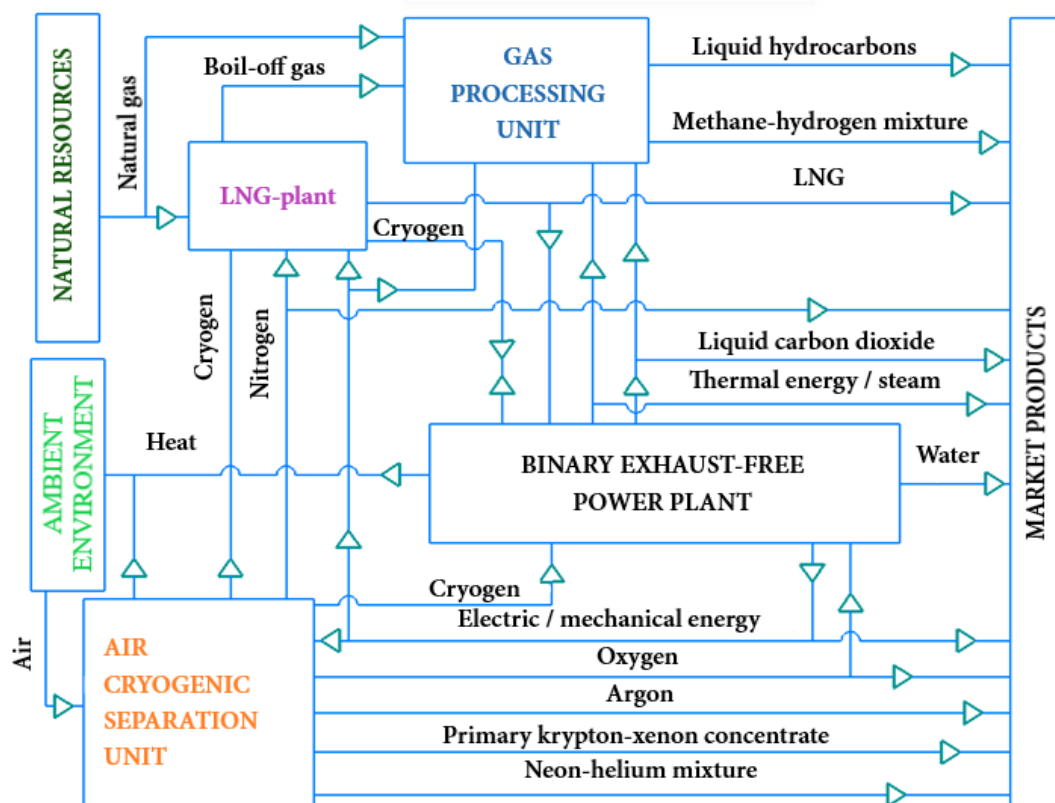
When the ambient environment temperature is increased, a deficiency of cold may be seen. Therefore, Figure 3 presents a potential solution to this problem with the use of heat pump (17).

#### **4. Using compressor-less combined cycle gas turbine in various energy-technological complexes**

Cold deficiency for recuperative compressor-less OCCGTU can also be reduced employing liquefied natural gas (LNG), when it is directly applied as the unit fuel. Here, profitability of using compressor-less OCCGTU, when upgrading coal CHPPs in Siberia and the Far East, should be noted. No gas pipelines and proximity of coal mines here is the main reason for using coal as a fuel for CHPPs in these Russian regions. LNG from existing and LNG-plants under construction in the Far East to such power stations can be delivered, similarly to coal, by railway in cryogenic tanks. Then, using compressor-less OCCGTU instead of coal-based CCGTU in this case solves both the problem of sequestering greenhouse gases from power-producing units and cogeneration boilers, and the problem of increasing the efficiency of electric generating equipment.

The following should be said further to the LNG subject matter. Worldwide natural gas consumption increases, and according to the IEA forecasts, the Asia-Pacific Region, which will account for almost 60% of the total growth in gas consumption by 2024, will define a major demand in the nearest years [22, 26]. And though our country is willing to sell pipeline gas to both China, the main gas consumer in the Asia-Pacific Region, and Europe, the LNG production becomes of importance for Russia, due to a global trend towards the greater use of LNG. Hence, of note are good prospects of employing OCCGTU in technological complexes of different types, including those created for chemical processing of gas and utilisation of carbon dioxide. Thus, integration of OCCGTU with the mentioned Novatek LNG-plant promises a cleaner production. Energy-related component in the "Arctic cascade" project is based on GTE with a relatively low efficiency and exhaust gases with CO<sub>2</sub>. The "Arctic cascade" project with compressor-less OCCGTU and integration of the power-producing unit thermodynamic cycle with the LNG-plant provide

considerable economic and environmental benefits. Figure 4 presents the flow diagram of such an energy-technological complex for liquefying and processing natural gas.



**Fig. 4. General flow diagram of an energy-technological complex for liquefying and processing natural gas [27]**

Low temperature of a cold source – the ambient environment is the prerequisite for a high efficiency of not only LNG-plant, but also OCCGTU with an air cryogenic separation unit, and interrelation between them in exchanging fluids and energy sets a very high standard for this efficiency. In this symbiosis, energetic OCCGTU's receiving the fuel in the liquid state from LNG-plants saves energy for its liquefaction and excludes the respective device in the OCCGTU structure (pos.10, Fig.3). Moreover, OCCGTU is capable of transmitting the required temperature and pressure to the unit of carbon dioxide gas and steam processing, and also oxygen to the ASU. Considerable ASU energy consumption is related to producing cold. The autonomous ASU therewith uses mostly cryogenic potential, and a major portion of "cold energy" is released to the ambient environment, whereas ASU in the considered energy-technological complex transfers along with nitrogen the "remnants of cold energy" to the LNG-plant, thus intensifying the specifics of the project, designed under low mean ambient temperature conditions [24].

Potential production of many diverse market products also affects the growth in the efficiency of integration project of the LNG-plant with OCCGTU. Use of energy, CO<sub>2</sub> and steam from OCCGTU and boil-off gas from LNG-plant creates conditions for one more opportunity for

such as technological symbiosis— production of liquid hydrocarbons and potential enrichment of natural gas with hydrogen.

From an environmental perspective, the use of a simple-cycle GTE [23, 24] as a source of energy for the LNG-plant is far from optimal option, what follows from the fact that gas equivalent of energy spent to liquefy gas amounts to about 10% of liquefied gas [26] with the respective CO<sub>2</sub> emission. In such a case, using OCCGTU, as a source of energy for the LNG-plant is justified.

In terms of profitability of employing compressor-less OCCGTUs, their use in the areas of underground coal mining is of a similar nature. Here, mandatory ventilation of mines releases into the atmosphere abundant methane, which greenhouse effect is 20 times greater than that of carbon dioxide. Using compressor-less OCCGTU enables sequestering of not only CO<sub>2</sub>, but also coal mine methane from ventilation drifts of mines [29]. In solving the problem with coal mine methane, it is possible to implement a hybrid energy-technological complex, when numerous ASUs with their energy from ventilation locations supply a “base” compressor-less OCCGTU with oxygen. Hybridization options of compressor-less OCCGTU go beyond it. Possible options of combined operation of compressor-less OCCGTU with renewable energy sources (RES) and nuclear power plants are under consideration, particularly, the use of high-pressure water electrolyzers instead of ASU to supply compressor-less OCCGTUs with oxygen [30].

The article examines no problems of using and burying carbon dioxide, to which numerous research and development publications [31-40] are devoted, however, it should be noted that the use of compressor-less OCCGTU can significantly affect the solutions suggested in these areas. The article describes the advantages of using compressor-less OCCGTU at different energy-technological facilities, first and foremost, in terms of the current vital task: reduction in man-made emissions of greenhouse gases. 180 countries, including Russia, which had signed the Paris climate agreement, are now faced with this task [41].

## 5. Conclusions

The main today's task of science and developers is to create priority projects of power-producing units, which ensure renewal of key funds of electric generating enterprises considering strict requirements imposed on greenhouse gas emissions. It is necessary to widen the searches for innovative concepts of creating high-efficiency power-producing units to the systems, that take into account energy production preparation, and energy generation, potential combination with process cycles and manufacture of chemical products, and utilisation of production waste as well.

The considered compressor-less CCGT unit is in compliance with the current requirements of high-efficiency eco-friendly power-producing units:

- generation of electric and thermal energy with the efficiency factor not lower than those of present-day CCGTUs, and also with a high CFU;
- no CO<sub>2</sub> emissions;
- use of this unit in various energy-technological complexes provides considerable economic and environmental benefits.

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